Continuous Second Order Sliding Mode Control
With On Line Tuned PID

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Abstract—This paper presents a robust control algorithm based on adaptive proportional-integral-derivative (PID) controller and sliding mode control (SMC) approach. In this paper, the drawbacks of SMC are compensated by adopting an adaptive PID continuous second order sliding mode control approach (APID2SMC) with PID sliding surface. The stability of the system is guaranteed in the sense of Lyapunov theorem. Numerical simulations using the dynamic model of DC motor with modeling uncertainties show that the proposed approach provides robustness for trajectory tracking performance under the occurrence of uncertainties. These simulation results compared to the results of conventional sliding mode control (CSMC) and PID control approaches indicate that the proposed method yields superior performance than the other controllers.

Index Terms— robust control, second order sliding mode control

I. INTRODUCTION

A sliding mode control (SMC) is one of the effective nonlinear robust control approaches since it provides system dynamics with an invariance property to uncertainties once the system dynamics are controlled in the sliding mode [4], [7]–[8], [11]–[12]. Under certain conditions, the SMC is robust with respect to system perturbation and external disturbance [4], [14]. However, this control strategy produces some drawbacks associated with large control chattering that may wear coupled mechanisms and excite undesirable high-frequency dynamics. Moreover, a priori knowledge of the upper bounds of the norm of the perturbation vector is required to obtain robustness and convergence. In order to overcome the drawbacks of SMC, the problem of perturbation estimation was investigated in [1] for a particular class of nonlinear uncertain systems. However, these problems remained mainly unsolved [9].

Several methods of chattering reduction have been reported. One approach [5] places a boundary layer around the switching surface such that the relay control is replaced by a saturation function. Another method [13] replaces a max–min-type control by a unit vector function. These approaches, however, provide no guarantee of convergence to the sliding mode and involve a tradeoff between chattering and robustness. In [6], an asymptotic observer in the control scheme can eliminate chattering despite the discontinuous control law. Both the boundary layer approach and the observer design assume that the unmodeled dynamics are completely unknown. In [2], the authors use the integral sliding method to overcome the drawback of chattering phenomenon. The authors in [10] attempted to reduce the well-known chattering problem in classical sliding mode control by presenting an innovative approach to adaptive fuzzy sliding mode control for a class of single input single output continuous nonlinear systems with unknown dynamics and bounded disturbances. In [14], a chattering free adaptive sliding mode controller is proposed for stabilizing a class of multi-input multi-output (MIMO) systems affected by both matched and mismatched types of uncertainties.

This paper discusses the design of a robust adaptive PID continuous second order sliding controller (APID2SMC) with PID sliding surface to achieve system robustness against parameter uncertainty, external load disturbance, and nonlinearities. In this approach, different from the second order sliding mode control presented in [3], an adaptive PID controller with on line tuned parameters replaces the equivalent control term which is not used because it needs exact full system dynamics knowledge and the discontinuous term is replaced by a PD control in terms of sliding surface dynamics. The proposed approach has some features such as a priori knowledge of the bounds of uncertainties is not required and the chattering phenomenon is eliminated without deterioration of the system robustness. The stability of the system is guaranteed in the sense of the Lyapunov stability theorem.

The proposed control algorithm is applied to a speed control of DC motor through simulations. The proposed approach is compared to the existing conventional controllers. The simulation results indicate that the control performance of the proposed control strategy is satisfactory and better than those of the existing conventional controllers. The remainder of this paper is organized as follows: section II presents the proposed adaptive PID second order sliding mode control. Section III presents the stability of the system in the sense of Lyapunov stability theorem. Section IV presents the simulation results, and section V summarizes the conclusions and contributions of the work.

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II. ADAPTIVE PID SECOND ORDER SLIDING MODE CONTROL

The main objective in this paper is to adopt an adaptive PID-based second order sliding mode control approach (APID2SMC) in which: (i) the equivalent control term in [3] is replaced by an online-tuned adaptive PID controller which results in more robustness, fast dynamic response and good steady state accuracy. (ii) the discontinuous control in [3] is replaced by a continuous proportional-derivative (PD) controller in terms of sliding surface dynamics which completely eliminates the requirement of a priori knowledge of the bounds of uncertainties and the chattering effects.

The dynamics of a single-input linear uncertain plant can be written as [3]:

\[
y(t) + a \cdot y(t) + b \cdot y(t) = c \cdot u(t) + d(t)
\]

where \( y(t) \) is the output, \( u(t) \) is the control input and \( d(t) \) represents the disturbances and un-modeled dynamics. \( a, b \) and \( c \) are plant parameters. In general, the system parameters involve a nominal part and some variations, therefore, in the presence of the model uncertainties and external disturbances, (1) can be written as follows:

\[
y(t) = -(a + \Delta a) \cdot y(t) - (b + \Delta b) \cdot y(t) + (c + \Delta c) \cdot u(t) + D(t)
\]

It is assumed that all uncertain elements can be lumped as follows:

\[
D(t) = -\Delta y(t) - \Delta b \cdot y(t) + \Delta c \cdot u(t) + D(t)
\]

The dynamics of the model is written as:

\[
y(t) = -a \cdot y(t) - b \cdot y(t) + c \cdot u(t) + D(t)
\]

The second order sliding PID surface in the space of tracking error can be defined as [3]:

\[
\sigma = s(t) + \alpha \cdot s(t) - e(t) + \lambda_1 \cdot e(t) + \lambda_2 \int e(\xi)d\xi
\]

where \( e(t) = y(t) - y_d(t) \) is the tracking error, in which \( y_d(t) \) is the desired output. \( \lambda_1, \lambda_2 \) and \( \alpha \) are positive constants.

The tracking control problem can be achieved by keeping the system trajectory on the sliding surface \( s(t) = 0 \). For this purpose, the proposed control algorithm is chosen as:

\[
u(t) = \frac{1}{c} \{ u_{PID}(t) + u_d(t) \}
\]

where \( u_{PID}(t) \) and \( u_d(t) \) are the adaptive PID control with online-tuned gains and the continuous PD control, respectively.

The adaptive PID control law \( u_{PID}(t) \) can be chosen as:

\[
u_{PID}(t) = -k_p \cdot e(t) - k_i \int e(\xi)d\xi - k_d \cdot e(t)
\]

where \( k_p, k_i \) and \( k_d \) are the adaptive proportional, integral and derivative gains to be tuned online.

The continuous proportional-derivative (PD) control law, \( u_a(t) \), can be chosen as:

\[
u_a(t) = -k_s \cdot s(t) - D_{est}(t)
\]

where \( k_s \) is a positive constant which is a design parameter and \( D_{est}(t) \) is the estimated perturbation.

III. LYAPUNOV STABILITY ANALYSIS

Lyapunov stability analysis is the most popular approach to prove and evaluate the stable convergence property of sliding mode control. Here, Lyapunov stability approach is employed to investigate the stability property of the proposed controller. Lyapunov function can be chosen as:

\[
V(t) = \frac{\alpha}{2} \cdot s^2(t) + \frac{1}{2} \cdot s^2(t) + \frac{1}{2} \cdot D^2(t) + \frac{1}{2} \cdot k_p^{-2} + \frac{1}{2} \cdot k_i^{-2}
\]

where \( \gamma_D, \gamma_p, \gamma_d \) and \( \gamma_i \) are positive constants which are design parameters and \( \tilde{D} \) is the lumped uncertainty estimation error.

The derivative of Lyapunov function is:

\[
\dot{V}(t) = \alpha \cdot s(t) + s(t) + \frac{1}{2} \cdot \dot{D} \cdot \tilde{D} + \frac{1}{2} \cdot k_p^{-2} \cdot k_p \cdot \dot{D} + \frac{1}{2} \cdot k_i^{-2} \cdot k_i \cdot \dot{D}
\]

Substituting (13) into (12) yields

\[
V(t) = \alpha \cdot s(t) + s(t) + \frac{1}{2} \cdot \dot{D} \cdot \tilde{D} + \frac{1}{2} \cdot k_p^{-2} \cdot k_p \cdot \dot{D} + \frac{1}{2} \cdot k_i^{-2} \cdot k_i \cdot \dot{D}
\]

By substituting (7) and (8) into (6), the proposed controller \( u(t) \) is chosen as:

\[
u(t) = \frac{1}{c} \{ u_{PID}(t) + u_d(t) \} = \frac{1}{c} [-k_p \cdot e(t) - k_i \int e(\xi)d\xi - k_d \cdot e(t) - k_s \cdot s(t) - D_{est}(t)]
\]

Substituting (15) into (14) yields:
\[ V(t) = s[\alpha s - \alpha s + s \cdot y_d - a y_d - b y_d + (\lambda_2 - \lambda_1) e (\lambda_2 - \lambda_1)] - k_s s - s D\varepsilon + s k_p \varepsilon + s k_i \int\varepsilon d \varepsilon + s D_{est} \varepsilon - sk \int\varepsilon d \varepsilon \]

Using (11), (21) can be manipulated as:

\[ V(t) = s[\alpha s - \alpha s + s \cdot y_d - a y_d - b y_d + (\lambda_2 - \lambda_1) e (\lambda_2 - \lambda_1)] - k_s s - s D\varepsilon + s k_p \varepsilon + s k_i \int\varepsilon d \varepsilon + s D_{est} \varepsilon - sk \int\varepsilon d \varepsilon \]

For \( V(t) < 0 \), the updated laws of the adaptive PID controller gains can be written as:

\[ k_p = \gamma_1 \varepsilon(t) s, \quad k_i = \gamma_2 \varepsilon(t) s \int\varepsilon d \varepsilon \quad (27) \]

The time evolution of the estimated uncertainty \( D_{est}(t) \) is chosen as:

\[ D_{est}(t) = \gamma_D s \quad (28) \]

Substituting (27) and (28) into (26) yields:
\begin{equation}
V(t) = (k + a + 1)s + s\left(\lambda_1 - a + 1 - \lambda_2 \right)e + a(\lambda_1 + \lambda_2 - b - k_p)e + (\lambda_2 - k_2)\int e^2 (s\lambda_1 - a \dot{y}_d - b \ddot{y}_d)\right)\right)
\end{equation}

For $V(t) < 0$, the following conditions must be satisfied:

\begin{equation}
k_d \geq \lambda_1 - a + 1, \quad k_p \geq \lambda_1 + \lambda_2 - b, \quad k_i \geq \lambda_2
\end{equation}

Substituting (30) into (29) yields

\begin{equation}
V(t) = -(k + a + 1)s + s\left(-\gamma_d - a \dot{y}_d - b \ddot{y}_d \right)
\end{equation}

From the above analysis the global asymptotic stability is guaranteed since $V(t) < 0$. From (6), (7), (8) and (28), the total proposed robust adaptive controller is expressed as:

\begin{equation}
u(t) = \frac{1}{c} \left[ u_{PID}(t) + u_o(t) \right] = \frac{1}{c} \left[ \gamma_d(t) - k_o \dot{e} - k_i \int e^2(t) \right] + \gamma(t)
\end{equation}

where the adaptive gains of PID term are defined in (27). The block diagram of the proposed control approach is depicted in Fig. 1.

IV. SIMULATION RESULTS

The proposed approach was tested for the speed control of the DC motor shown in Fig. 2 to validate the feasibility and effectiveness of the presented control algorithm.

\begin{equation}
w(t) = -aw(t) - bw(t) + cw(t) + D(t)
\end{equation}

where $w$ represents speed, $a = \left( \frac{B}{J} + \frac{R_a}{L_a} \right)$, $b = \left( \frac{BR_a + k_b}{JL_a} \right)$ and $c = \frac{k_i}{JL_a}$. $u(t) = E_a$ represents the motor armature voltage, $R_a = 1.6\Omega$, $L_a = 0.1H$ represent the armature coil resistance and inductance, respectively, $J = 0.1N\cdot m/\text{rad/s}^2$ is the moment of inertia, $B = 0.5N\cdot m/\text{rad/s}$ represents coefficient of viscous friction, $k_i = 0.1N\cdot m/\text{A}$ represents the torque constant and $k_b = 0.1V/\text{rad/s}$. represents the back emf constant. All simulations are carried out using MATLAB R2009b.

In order to show the effectiveness and robustness of the proposed approach, large severe uncertainties $D(t)$ are occurred which contains:

1. A step external load disturbance torque of $5N\cdot m$ is suddenly applied at the motor shaft at $t = 1.0$ s. and is shown in Fig. 3.

2. An unpredicted variation of the armature resistance $R_a$ and coefficient of viscous friction $B$ occurred at $t = 1.0$ s. The abrupt changes of $R_a$ and $B$ are shown in Fig. 4.
Simulations are performed in the case of set-point tracking in which the command desired motor speed is set equal to 500 rpm. For comparison, conventional PID controller and conventional sliding mode control (CSMC) are used.

The parameters of the proposed APID2SMC are set as: \( \alpha = 50, \quad \lambda_1 = 2, \quad \lambda_2 = 1, \quad k = 20, \quad \gamma_D = 10, \quad \gamma_p = 0.5, \quad \gamma_i = 0.08 \) and \( \gamma_d = 0.01 \).

Fig. 5 shows the DC motor speed response profile. The time history of the sliding surface \( s(t) \) for CSMC is depicted in Fig. 6. The phase portrait trajectory for CSMC is shown in Fig. 7. The time history of the sliding surface \( s(t) \) for APID2SMC is depicted in Fig. 8. The phase portrait trajectory for APID2SMC is shown in Fig. 9. The trajectory of the adaptive proportional gain \( \hat{k}_p \), adaptive integral gain \( \hat{k}_i \) and adaptive derivative gain \( \hat{k}_d \) defined in (27) for APID2SMC is shown in Fig. 10.
The presented paper has presented an adaptive PID continuous second order sliding mode control methodology (APIDSMC) with PID sliding surface to achieve system robustness against parameter uncertainty and external load disturbance. In this approach, an adaptive PID controller with parameters being updated online replaces the equivalent control term and the discontinuous term is replaced by a continuous PD control in terms of sliding surface dynamics.

The proposed control algorithm has some features such as a priori knowledge of the bounds of uncertainties is not required and the chattering phenomenon is eliminated without deteriorating the system robustness. The closed-loop system is globally stable in the Lyapunov sense and the system output can track the desired output asymptotically in the presence modeling uncertainties and disturbances. Simulation results have showed that the performance of the proposed controller is better than the conventional approaches and has superior tracking performance with robust characteristics to conventional methods.

**REFERENCES**


